

Chapter 1:

$$F = ma$$

$$1 \text{ J} = 1 \text{ N}\cdot\text{m}$$

$$\text{Density:} \quad \rho = \frac{m}{V} \quad (\text{kg/m}^3)$$

$$v = \frac{V}{m} = \frac{1}{\rho}$$

$$\text{Specific gravity:} \quad \text{SG} = \frac{\rho}{\rho_{\text{H}_2\text{O}}}$$

$$\text{Specific weight:} \quad \gamma_s = \rho g \quad (\text{N/m}^3)$$

$$T = a + bP$$

$$T(\text{K}) = T(^{\circ}\text{C}) + 273.15$$

$$T(\text{R}) = T(^{\circ}\text{F}) + 459.67$$

$$T(\text{R}) = 1.8T(\text{K})$$

$$T(^{\circ}\text{F}) = 1.8T(^{\circ}\text{C}) + 32$$

$$\Delta T(\text{K}) = \Delta T(^{\circ}\text{C})$$

$$\Delta T(\text{R}) = \Delta T(^{\circ}\text{F})$$

$$P_{\text{gage}} = P_{\text{abs}} - P_{\text{atm}}$$

$$P_{\text{vac}} = P_{\text{atm}} - P_{\text{abs}}$$

$$\Delta P = P_2 - P_1 = -\rho g \Delta z = -\gamma_s \Delta z$$

$$P_{\text{below}} = P_{\text{above}} + \rho g |\Delta z| = P_{\text{above}} + \gamma_s |\Delta z|$$

$$P = P_{\text{atm}} + \rho gh \quad \text{or} \quad P_{\text{gage}} = \rho gh$$

$$\frac{dP}{dz} = -\rho g$$

$$\Delta P = P_2 - P_1 = -\int_1^2 \rho g dz$$

$$P_1 = P_2 \quad \rightarrow \quad \frac{F_1}{A_1} = \frac{F_2}{A_2} \quad \rightarrow \quad \frac{F_2}{F_1} = \frac{A_2}{A_1}$$

$$P_{\text{atm}} = \rho gh \quad P_2 = P_{\text{atm}} + \rho gh$$

$$P_1 + \rho_1 g(a + h) - \rho_2 gh - \rho_1 ga = P_2$$

$$P_1 - P_2 = (\rho_2 - \rho_1)gh$$

Chapter 2:

$$e = \frac{E}{m} \quad (\text{kJ/kg})$$

$$\text{KE} = m \frac{V^2}{2} \quad (\text{kJ}) \quad \text{ke} = \frac{V^2}{2} \quad (\text{kJ/kg})$$

$$\text{PE} = mgz \quad (\text{kJ}) \quad \text{pe} = gz \quad (\text{kJ/kg})$$

$$E = U + \text{KE} + \text{PE} = U + m \frac{V^2}{2} + mgz \quad (\text{kJ})$$

$$e = u + \text{ke} + \text{pe} = u + \frac{V^2}{2} + gz \quad (\text{kJ/kg})$$

$$\text{Mass flow rate:} \quad \dot{m} = \rho \dot{V} = \rho A_c V_{\text{avg}} \quad (\text{kg/s})$$

$$\text{Energy flow rate:} \quad \dot{E} = \dot{m}e \quad (\text{kJ/s or kW})$$

$$e_{\text{mech}} = \frac{P}{\rho} + \frac{V^2}{2} + gz$$

$$\dot{E}_{\text{mech}} = \dot{m}e_{\text{mech}} = \dot{m} \left(\frac{P}{\rho} + \frac{V^2}{2} + gz \right)$$

$$\Delta e_{\text{mech}} = \frac{P_2 - P_1}{\rho} + \frac{V_2^2 - V_1^2}{2} + g(z_2 - z_1) \quad (\text{kJ/kg})$$

$$\Delta \dot{E}_{\text{mech}} = \dot{m} \Delta e_{\text{mech}} = \dot{m} \left(\frac{P_2 - P_1}{\rho} + \frac{V_2^2 - V_1^2}{2} + g(z_2 - z_1) \right) \quad (\text{kW})$$

$$q = \frac{Q}{m} \quad (\text{kJ/kg})$$

$$Q = \int_{t_1}^{t_2} \dot{Q} dt \quad (\text{kJ})$$

$$Q = \dot{Q} \Delta t \quad (\text{kJ})$$

$$w = \frac{W}{m} \quad (\text{kJ/kg})$$

$$\int_1^2 dV = V_2 - V_1 = \Delta V$$

$$\int_1^2 \delta W = W_{12} \quad (\text{not } \Delta W)$$

$$\dot{W}_e = \mathbf{VI} \quad (\text{W})$$

$$W_e = \int_1^2 \mathbf{VI} dt \quad (\text{kJ})$$

$$W_e = \mathbf{VI} \Delta t \quad (\text{kJ})$$

$$W = Fs \quad (\text{kJ})$$

$$W = \int_1^2 F ds \quad (\text{kJ})$$

$$\mathbf{T} = Fr \rightarrow F = \frac{\mathbf{T}}{r}$$

$$s = (2\pi r)n$$

$$W_{\text{sh}} = Fs = \left(\frac{T}{r}\right)(2\pi rn) = 2\pi nT \quad (\text{kJ})$$

$$\dot{W}_{\text{sh}} = 2\pi \dot{n}T \quad (\text{kW})$$

$$\delta W_{\text{spring}} = F dx$$

$$F = kx \quad (\text{kN})$$

$$W_{\text{spring}} = \frac{1}{2}k(x_2^2 - x_1^2) \quad (\text{kJ})$$

$$W_{\text{elastic}} = \int_1^2 F dx = \int_1^2 \sigma_n A dx \quad (\text{kJ})$$

$$W_{\text{surface}} = \int_1^2 \sigma_s dA \quad (\text{kJ})$$

$$\Delta E_{\text{system}} = E_{\text{final}} - E_{\text{initial}} = E_2 - E_1$$

$$\Delta E = \Delta U + \Delta \text{KE} + \Delta \text{PE}$$

$$\Delta U = m(u_2 - u_1)$$

$$\Delta \text{KE} = \frac{1}{2}m(V_2^2 - V_1^2)$$

$$\Delta \text{PE} = mg(z_2 - z_1)$$

$$E_{\text{in}} - E_{\text{out}} = (Q_{\text{in}} - Q_{\text{out}}) + (W_{\text{in}} - W_{\text{out}}) + (E_{\text{mass,in}} - E_{\text{mass,out}}) = \Delta E_{\text{system}}$$

$$\underbrace{E_{\text{in}} - E_{\text{out}}}_{\substack{\text{Net energy transfer} \\ \text{by heat, work, and mass}}} = \underbrace{\Delta E_{\text{system}}}_{\substack{\text{Change in internal, kinetic,} \\ \text{potential, etc., energies}}} \quad (\text{kJ})$$

$$\underbrace{\dot{E}_{\text{in}} - \dot{E}_{\text{out}}}_{\substack{\text{Rate of net energy transfer} \\ \text{by heat, work, and mass}}} = \underbrace{dE_{\text{system}}/dt}_{\substack{\text{Rate of change in internal,} \\ \text{kinetic, potential, etc., energies}}} \quad (\text{kW})$$

$$Q = \dot{Q} \Delta t, \quad W = \dot{W} \Delta t, \quad \text{and} \quad \Delta E = (dE/dt) \Delta t \quad (\text{kJ})$$

$$e_{\text{in}} - e_{\text{out}} = \Delta e_{\text{system}} \quad (\text{kJ/kg})$$

$$\delta E_{\text{in}} - \delta E_{\text{out}} = dE_{\text{system}} \quad \text{or} \quad \delta e_{\text{in}} - \delta e_{\text{out}} = de_{\text{system}}$$

$$W_{\text{net,out}} = Q_{\text{net,in}} \quad \text{or} \quad \dot{W}_{\text{net,out}} = \dot{Q}_{\text{net,in}} \quad (\text{for a cycle})$$

$$\text{Efficiency} = \frac{\text{Desired output}}{\text{Required input}}$$

$$\eta_{\text{combustion}} = \frac{Q}{\text{HV}} = \frac{\text{Amount of heat released during combustion}}{\text{Heating value of the fuel burned}}$$

$$\eta_{\text{overall}} = \eta_{\text{combustion}} \eta_{\text{thermal}} \eta_{\text{generator}} = \frac{\dot{W}_{\text{net,electric}}}{\text{HHV} \times \dot{m}_{\text{fuel}}}$$

$$\eta_{\text{mech}} = \frac{\text{Mechanical energy output}}{\text{Mechanical energy input}} = \frac{E_{\text{mech,out}}}{E_{\text{mech,in}}} = 1 - \frac{E_{\text{mech,loss}}}{E_{\text{mech,in}}}$$

$$\eta_{\text{pump}} = \frac{\text{Mechanical energy increase of the fluid}}{\text{Mechanical energy input}} = \frac{\Delta \dot{E}_{\text{mech,fluid}}}{\dot{W}_{\text{shaft,in}}} = \frac{\dot{W}_{\text{pump},u}}{\dot{W}_{\text{pump}}}$$

$$\eta_{\text{turbine}} = \frac{\text{Mechanical energy output}}{\text{Mechanical energy decrease of the fluid}} = \frac{\dot{W}_{\text{shaft,out}}}{|\Delta \dot{E}_{\text{mech,fluid}}|} = \frac{\dot{W}_{\text{turbine}}}{\dot{W}_{\text{turbine},e}}$$

$$\text{Motor:} \quad \eta_{\text{motor}} = \frac{\text{Mechanical power output}}{\text{Electric power input}} = \frac{\dot{W}_{\text{shaft,out}}}{\dot{W}_{\text{elect,in}}}$$

$$\text{Generator:} \quad \eta_{\text{generator}} = \frac{\text{Electric power output}}{\text{Mechanical power input}} = \frac{\dot{W}_{\text{elect,out}}}{\dot{W}_{\text{shaft,in}}}$$

$$\eta_{\text{pump-motor}} = \eta_{\text{pump}} \eta_{\text{motor}} = \frac{\dot{W}_{\text{pump},u}}{\dot{W}_{\text{elect,in}}} = \frac{\Delta \dot{E}_{\text{mech,fluid}}}{\dot{W}_{\text{elect,in}}}$$

$$\eta_{\text{turbine-gen}} = \eta_{\text{turbine}} \eta_{\text{generator}} = \frac{\dot{W}_{\text{elect,out}}}{\dot{W}_{\text{turbine},e}} = \frac{\dot{W}_{\text{elect,out}}}{|\Delta \dot{E}_{\text{mech,fluid}}|}$$

Chapter 3:

$$h = u + Pv \quad (\text{kJ/kg})$$

$$H = U + PV \quad (\text{kJ})$$

$$x = \frac{m_{\text{vapor}}}{m_{\text{total}}}$$

$$v_{\text{avg}} = v_f + xv_{fg} \quad (\text{m}^3/\text{kg})$$

$$x = \frac{v_{\text{avg}} - v_f}{v_{fg}}$$

$$u_{\text{avg}} = u_f + xu_{fg} \quad (\text{kJ/kg})$$

$$h_{\text{avg}} = h_f + xh_{fg} \quad (\text{kJ/kg})$$

$$y \cong y_{f@T}$$

$$h \cong h_{f@T} + v_{f@T}(P - P_{\text{sat}@T})$$

$$Pv = RT$$

$$m = MN \quad (\text{kg})$$

$$V = m v \longrightarrow PV = mRT$$

$$mR = (MN)R = NR_u \longrightarrow PV = NR_u T$$

$$V = N\bar{v} \longrightarrow P\bar{v} = R_u T$$

$$\frac{P_1 V_1}{T_1} = \frac{P_2 V_2}{T_2}$$

Chapter 4:

(1) General $W_b = \int_1^2 P dV$

(2) Isobaric process
 $W_b = P_0(V_2 - V_1)$ ($P_1 = P_2 = P_0 = \text{constant}$)

(3) Polytropic process
 $W_b = \frac{P_2V_2 - P_1V_1}{1 - n}$ ($n \neq 1$) ($PV^n = \text{constant}$)

(4) Isothermal process of an ideal gas
 $W_b = P_1V_1 \ln \frac{V_2}{V_1}$
 $= mRT_0 \ln \frac{V_2}{V_1}$ ($PV = mRT_0 = \text{constant}$)

For a closed system $Q - W = \Delta U + \Delta KE + \Delta PE$
 $W = W_{\text{other}} + W_b$

For a constant pressure process $Q - W_{\text{other}} = \Delta H + \Delta KE + \Delta PE$

$$\Delta u = u_2 - u_1 = \int_1^2 c_v(T) dT \equiv c_{v,\text{avg}}(T_2 - T_1)$$

$$\Delta h = h_2 - h_1 = \int_1^2 c_p(T) dT \equiv c_{p,\text{avg}}(T_2 - T_1)$$

For an ideal gas $c_p = c_v + R$

Specific heat ratio $k = \frac{c_p}{c_v}$

For incompressible substances (liquids and solids) $c_p = c_v = c$

The Δu and Δh of incompressible substances $\Delta u = \int_1^2 c(T) dT \equiv c_{\text{avg}}(T_2 - T_1)$
 $\Delta h = \Delta u + v\Delta P$

Chapter 5

$$m_{\text{in}} - m_{\text{out}} = \Delta m_{\text{system}} \quad \text{and} \quad \dot{m}_{\text{in}} - \dot{m}_{\text{out}} = dm_{\text{system}}/dt$$

$$\dot{m} = \rho VA$$

$$\dot{V} = VA = \dot{m}/\rho$$

$$\theta = h + ke + pe = h + \frac{V^2}{2} + gz$$

$$\underbrace{E_{\text{in}} - E_{\text{out}}}_{\text{Rate of net energy transfer by heat, work, and mass}} = \underbrace{dE_{\text{system}}/dt}_{\text{Rate of change in internal, kinetic, potential, etc., energies}}$$

$$\sum_{\text{in}} \dot{m} = \sum_{\text{out}} \dot{m}$$

$$\dot{Q} - \dot{W} = \sum_{\text{out}} \dot{m} \underbrace{\left(h + \frac{V^2}{2} + gz \right)}_{\text{for each exit}} - \sum_{\text{in}} \dot{m} \underbrace{\left(h + \frac{V^2}{2} + gz \right)}_{\text{for each inlet}}$$

$$\dot{m}_1 = \dot{m}_2 \longrightarrow \frac{1}{v_1} V_1 A_1 = \frac{1}{v_2} V_2 A_2$$

$$\dot{Q} - \dot{W} = \dot{m} \left[h_2 - h_1 + \frac{V_2^2 - V_1^2}{2} + g(z_2 - z_1) \right]$$

General Steady State Energy Equation:

$$0 = \frac{\dot{Q}_{\text{cv}}}{\dot{m}} - \frac{\dot{W}_{\text{cv}}}{\dot{m}} + (h_1 - h_2) + \frac{(V_1^2 - V_2^2)}{2} + g(z_1 - z_2)$$

General Nozzle Eq: $\dot{W}_{\text{cv}} = \dot{m}(h_1 - h_2)$

General Turbine Eq: $\frac{\dot{W}_{\text{out}}}{\dot{m}} = (h_1 - h_2) + \left(\frac{V_1^2 - V_2^2}{2} \right)$

General Compressor and Pump Eq: $\dot{W}_{\text{cv}} = \dot{m}(h_1 - h_2)$

General Heat Exchanger Eq.: $0 = \sum_i \dot{m}_i h_i - \sum_e \dot{m}_e h_e$

General Throttling and Valves Eq.: $h_1 = h_2$

Chapter 6

$$\eta_{\text{th}} = \frac{W_{\text{net,out}}}{Q_H} \quad \text{or} \quad \eta_{\text{th}} = 1 - \frac{Q_L}{Q_H}$$

$$\text{COP}_R = \frac{Q_L}{Q_H - Q_L} = \frac{1}{Q_H/Q_L - 1}$$

$$\text{COP}_{\text{HP}} = \frac{Q_H}{Q_H - Q_L} = \frac{1}{1 - Q_L/Q_H}$$

$$\eta_{\text{th,rev}} = 1 - \frac{T_L}{T_H} \quad \text{COP}_{R,\text{rev}} = \frac{1}{T_H/T_L - 1} \quad \text{COP}_{\text{HP,rev}} = \frac{1}{1 - T_L/T_H}$$

$$\left(\frac{Q_H}{Q_L}\right)_{\text{rev}} = \frac{T_H}{T_L}$$

Chapter 7

$$dS = \left(\frac{\delta Q}{T} \right)_{\text{int rev}} \quad (\text{kJ/K}) \quad \Delta S = \frac{Q}{T_0} \quad (\text{kJ/K})$$

$$\Delta S_{\text{sys}} = S_2 - S_1 = \int_1^2 \frac{\delta Q}{T} + S_{\text{gen}}$$

$$\Delta S_{\text{isolated}} \geq 0$$

$$S_{\text{gen}} = \Delta S_{\text{total}} = \Delta S_{\text{sys}} + \Delta S_{\text{surr}} \geq 0$$

$$\Delta S = m\Delta s = m(s_2 - s_1) \quad (\text{kJ/K})$$

$$\text{Isentropic process:} \quad \Delta s = 0 \quad \text{or} \quad s_2 = s_1 \quad (\text{kJ/kg}\cdot\text{K})$$

$$Q_{\text{int rev}} = \int_1^2 T dS \quad (\text{kJ})$$

$$Q_{\text{int rev}} = T_0 \Delta S \quad (\text{kJ}) \quad q_{\text{int rev}} = T_0 \Delta s \quad (\text{kJ/kg})$$

$$T ds = du + P dv \quad (\text{kJ/kg})$$

$$\left. \begin{array}{l} h = u + Pv \quad \longrightarrow \quad dh = du + P dv + v dP \\ (\text{Eq. 7-23}) \quad \longrightarrow \quad T ds = du + P dv \end{array} \right\} T ds = dh - v dP$$

$$ds = \frac{du}{T} + \frac{P dv}{T} \quad ds = \frac{dh}{T} - \frac{v dP}{T}$$

$$\text{Liquids, solids:} \quad s_2 - s_1 = \int_1^2 c(T) \frac{dT}{T} \cong c_{\text{avg}} \ln \frac{T_2}{T_1} \quad (\text{kJ/kg}\cdot\text{K})$$

$$\text{Isentropic:} \quad s_2 - s_1 = c_{\text{avg}} \ln \frac{T_2}{T_1} = 0 \quad \longrightarrow \quad T_2 = T_1$$

For an ideal gas

$$ds = c_v \frac{dT}{T} + R \frac{dv}{v} \quad s_2 - s_1 = \int_1^2 c_v(T) \frac{dT}{T} + R \ln \frac{v_2}{v_1} \quad s_2 - s_1 = \int_1^2 c_p(T) \frac{dT}{T} - R \ln \frac{P_2}{P_1}$$

For an ideal gas assuming constant heat capacities

$$s_2 - s_1 = c_{v,\text{avg}} \ln \frac{T_2}{T_1} + R \ln \frac{v_2}{v_1} \quad (\text{kJ/kg}\cdot\text{K})$$

$$s_2 - s_1 = c_{p,\text{avg}} \ln \frac{T_2}{T_1} - R \ln \frac{P_2}{P_1} \quad (\text{kJ/kg}\cdot\text{K})$$

For an ideal gas with variable heat capacities

$$s_2 - s_1 = s_2^\circ - s_1^\circ - R \ln \frac{P_2}{P_1} \quad (\text{kJ/kg}\cdot\text{K})$$

Isentropic process of ideal gases

$$\left(\frac{T_2}{T_1}\right)_{s=\text{const.}} = \left(\frac{v_1}{v_2}\right)^{k-1} \quad (\text{ideal gas})$$

$$\left(\frac{T_2}{T_1}\right)_{s=\text{const.}} = \left(\frac{P_2}{P_1}\right)^{(k-1)/k} \quad (\text{ideal gas})$$

$$\left(\frac{P_2}{P_1}\right)_{s=\text{const.}} = \left(\frac{v_1}{v_2}\right)^k \quad (\text{ideal gas})$$

$$s_2^\circ = s_1^\circ + R \ln \frac{P_2}{P_1}$$

$$\left(\frac{P_2}{P_1}\right)_{s=\text{const.}} = \frac{P_{r2}}{P_{r1}} \quad \left(\frac{v_2}{v_1}\right)_{s=\text{const.}} = \frac{v_{r2}}{v_{r1}}$$

Reversible steady flow work

$$w_{\text{rev}} = - \int_1^2 v dP - \Delta \text{ke} - \Delta \text{pe} \quad (\text{kJ/kg})$$

$$w_{\text{rev,in}} = \int_1^2 v dP + \Delta \text{ke} + \Delta \text{pe}$$

$$w_{\text{rev}} = -v(P_2 - P_1) - \Delta \text{ke} - \Delta \text{pe} \quad (\text{kJ/kg})$$

Compressor work

$$\text{Isentropic} \quad w_{\text{comp,in}} = \frac{kR(T_2 - T_1)}{k - 1} = \frac{kRT_1}{k - 1} \left[\left(\frac{P_2}{P_1} \right)^{(k-1)/k} - 1 \right]$$

$$\text{Polytropic} \quad w_{\text{comp,in}} = \frac{nR(T_2 - T_1)}{n - 1} = \frac{nRT_1}{n - 1} \left[\left(\frac{P_2}{P_1} \right)^{(n-1)/n} - 1 \right]$$

$$\text{Isothermal} \quad w_{\text{comp,in}} = RT \ln \frac{P_2}{P_1}$$

$$P_x = (P_1 P_2)^{1/2} \quad \text{or} \quad \frac{P_x}{P_1} = \frac{P_2}{P_x}$$

Isentropic efficiency of turbines

$$\eta_T = \frac{\text{Actual turbine work}}{\text{Isentropic turbine work}} = \frac{w_a}{w_s} \quad \eta_T \equiv \frac{h_1 - h_{2a}}{h_1 - h_{2s}}$$

Isentropic efficiency of compressors and pumps

$$\eta_C = \frac{\text{Isentropic compressor work}}{\text{Actual compressor work}} = \frac{w_s}{w_a} \quad \eta_C \equiv \frac{h_{2s} - h_1}{h_{2a} - h_1}$$

Isentropic efficiency of nozzle

$$\eta_N = \frac{\text{Actual KE at nozzle exit}}{\text{Isentropic KE at nozzle exit}} = \frac{V_{2a}^2}{V_{2s}^2} \quad \eta_N \equiv \frac{h_1 - h_{2a}}{h_1 - h_{2s}}$$

$$\Delta S_{\text{system}} = S_{\text{final}} - S_{\text{initial}} = S_2 - S_1$$

$$\text{Entropy transfer by heat transfer:} \quad S_{\text{heat}} = \frac{Q}{T} \quad (T = \text{constant})$$

$$\text{Entropy transfer by work:} \quad S_{\text{work}} = 0$$

$$\text{Entropy transfer by mass flow:} \quad S_{\text{mass}} = ms$$

$$\underbrace{S_{\text{in}} - S_{\text{out}}}_{\text{Net entropy transfer by heat and mass}} + \underbrace{S_{\text{gen}}}_{\text{Entropy generation}} = \underbrace{\Delta S_{\text{system}}}_{\text{Change in entropy}} \quad (\text{kJ/K})$$

$$\text{Closed system:} \quad \sum \frac{Q_k}{T_k} + S_{\text{gen}} = \Delta S_{\text{system}} = S_2 - S_1 \quad (\text{kJ/K})$$

$$\text{Adiabatic closed system:} \quad S_{\text{gen}} = \Delta S_{\text{adiabatic system}}$$

$$\sum \frac{Q_k}{T_k} + \sum m_i s_i - \sum m_e s_e + S_{\text{gen}} = (S_2 - S_1)_{\text{CV}} \quad (\text{kJ/K})$$

$$\sum \frac{\dot{Q}_k}{T_k} + \sum \dot{m}_i s_i - \sum \dot{m}_e s_e + \dot{S}_{\text{gen}} = dS_{\text{CV}}/dt \quad (\text{kW/K})$$

$$\text{Steady-flow:} \quad \dot{S}_{\text{gen}} = \sum \dot{m}_e s_e - \sum \dot{m}_i s_i - \sum \frac{\dot{Q}_k}{T_k}$$

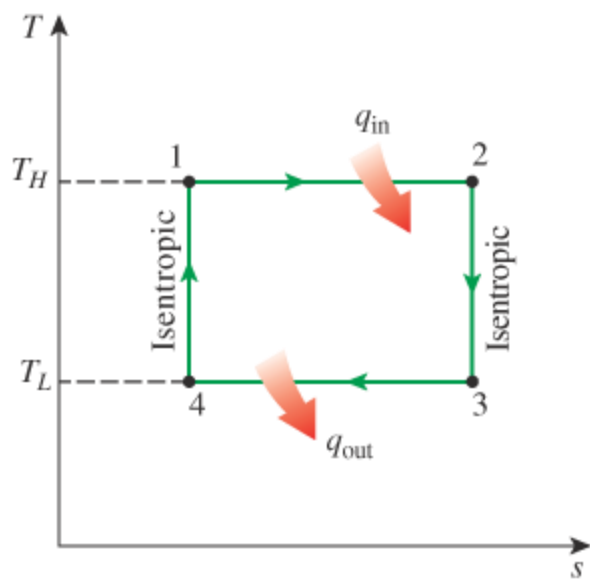
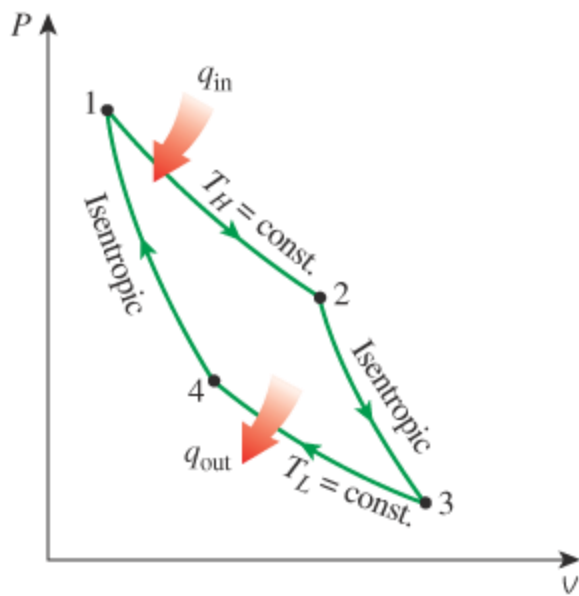
$$\text{Steady-flow, single-stream:} \quad \dot{S}_{\text{gen}} = \dot{m}(s_e - s_i) - \sum \frac{\dot{Q}_k}{T_k}$$

$$\text{Steady-flow, single-stream, adiabatic:} \quad \dot{S}_{\text{gen}} = \dot{m}(s_e - s_i)$$

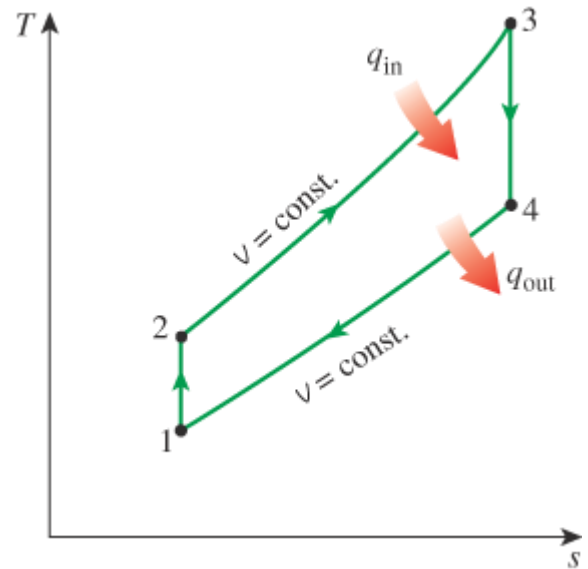
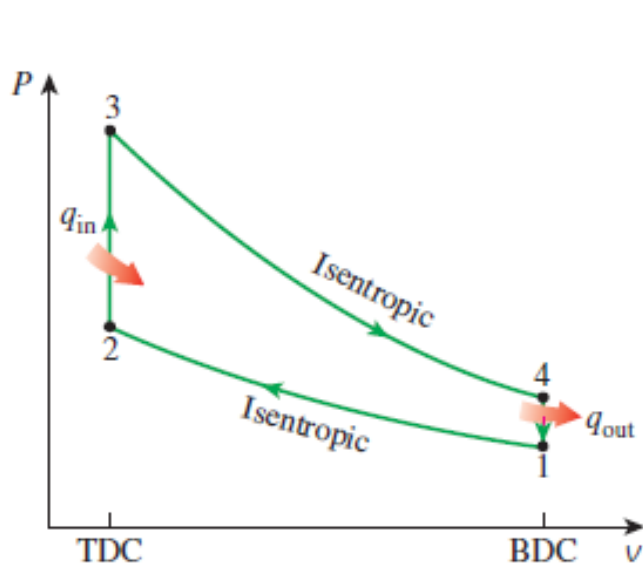
Chapter 9

$$\eta_{\text{th,Carnot}} = 1 - \frac{T_L}{T_H} \quad \eta_{\text{th}} = \frac{W_{\text{net}}}{Q_{\text{in}}} \quad \text{or} \quad \eta_{\text{th}} = \frac{W_{\text{net}}}{q_{\text{in}}}$$

Carnot cycle



Otto cycle



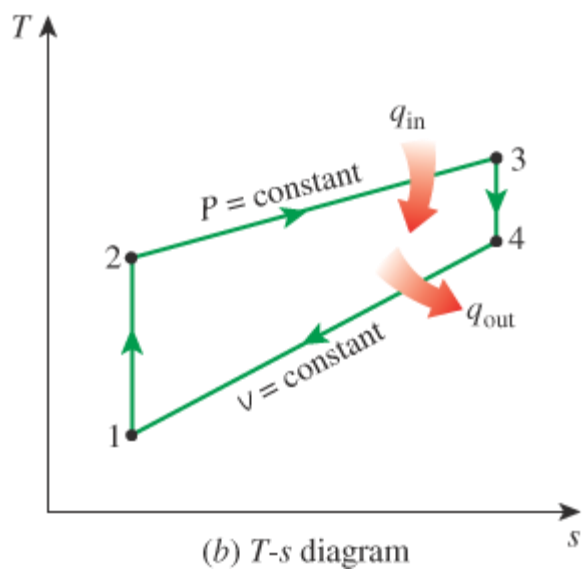
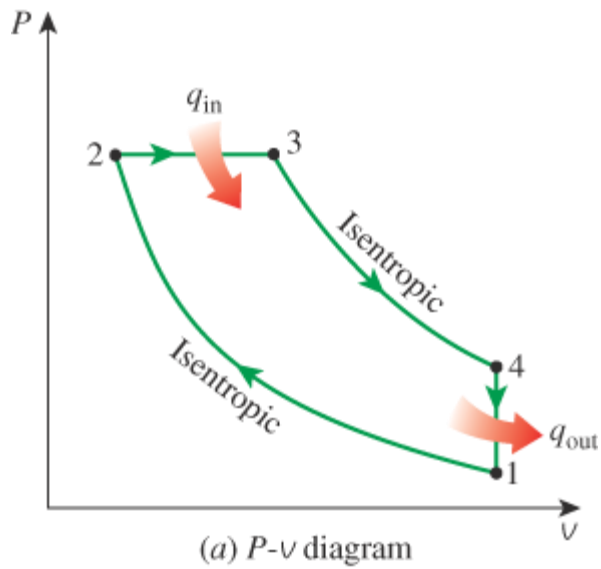
$$r = \frac{V_{\max}}{V_{\min}} = \frac{V_{\text{BDC}}}{V_{\text{TDC}}}$$

$$\text{MEP} = \frac{W_{\text{net}}}{V_{\max} - V_{\min}} = \frac{W_{\text{net}}}{v_{\max} - v_{\min}} \quad (\text{kPa})$$

Assuming constant heat capacities

$$\eta_{\text{th, Otto}} = 1 - \frac{1}{r^{k-1}}$$

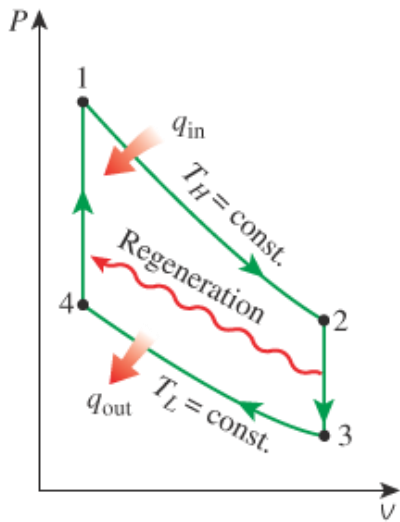
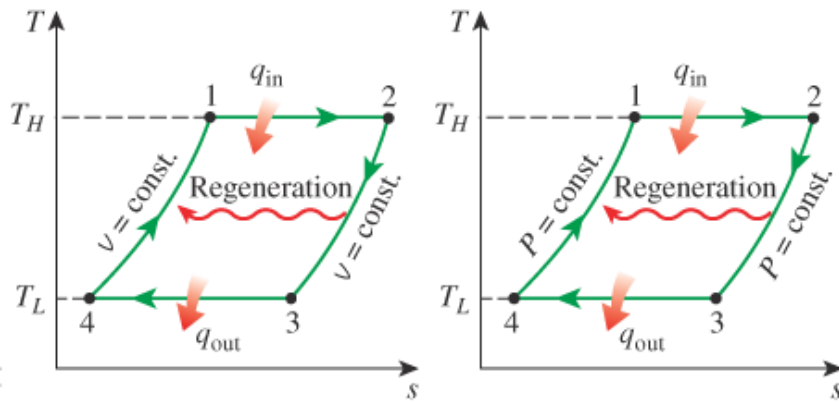
Diesel cycle



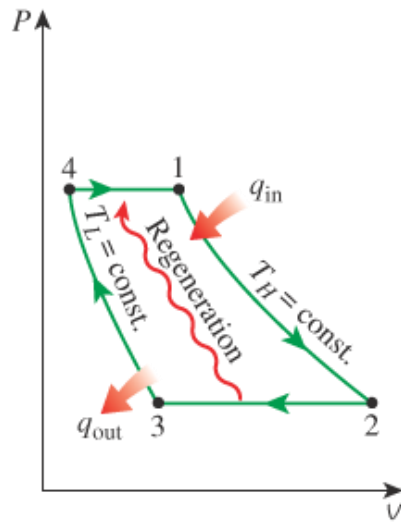
Assuming constant heat capacities

$$\eta_{\text{th,Diesel}} = 1 - \frac{1}{r^{k-1}} \left[\frac{r_c^k - 1}{k(r_c - 1)} \right] \quad r_c = \frac{V_3}{V_2} = \frac{v_3}{v_2}$$

Stirling and Ericsson cycles



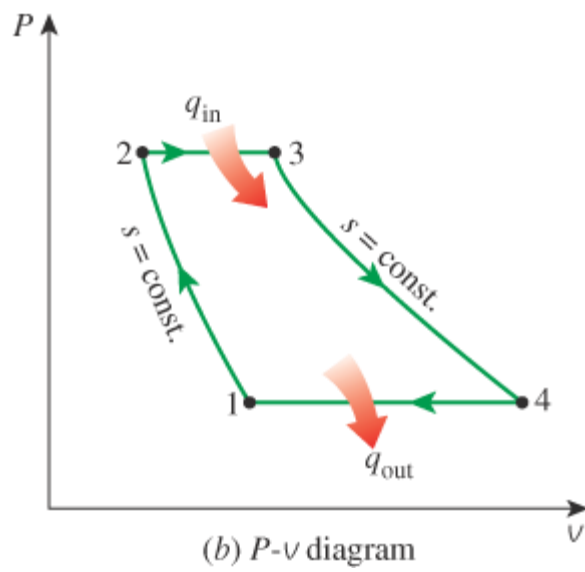
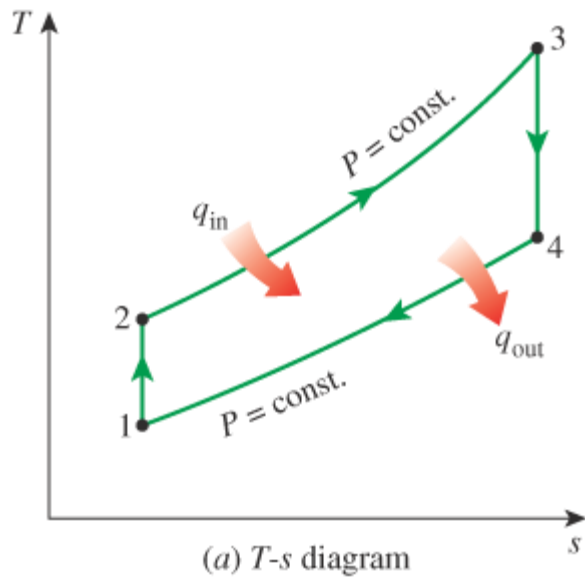
(b) Stirling cycle



(c) Ericsson cycle

$$\eta_{\text{th,Stirling}} = \eta_{\text{th,Ericsson}} = \eta_{\text{th,Carnot}} = 1 - \frac{T_L}{T_H}$$

Brayton cycle

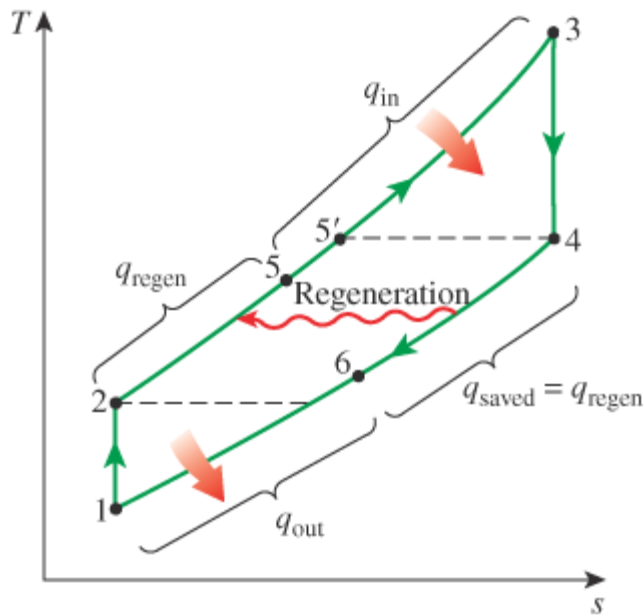


Assuming constant heat capacities

$$\eta_{th,Brayton} = 1 - \frac{1}{r_p^{(k-1)/k}}$$

$$r_p = \frac{P_2}{P_1}$$

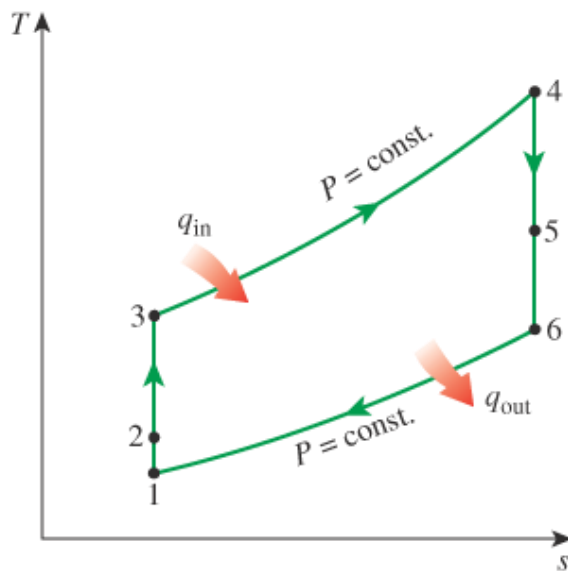
Brayton cycle with regeneration



$$\epsilon = \frac{q_{\text{regen,act}}}{q_{\text{regen,max}}} = \frac{h_5 - h_2}{h_4 - h_2}$$

$$\eta_{\text{th,regen}} = 1 - \left(\frac{T_1}{T_3}\right)(r_p)^{(k-1)/k}$$

Ideal jet-propulsion cycle



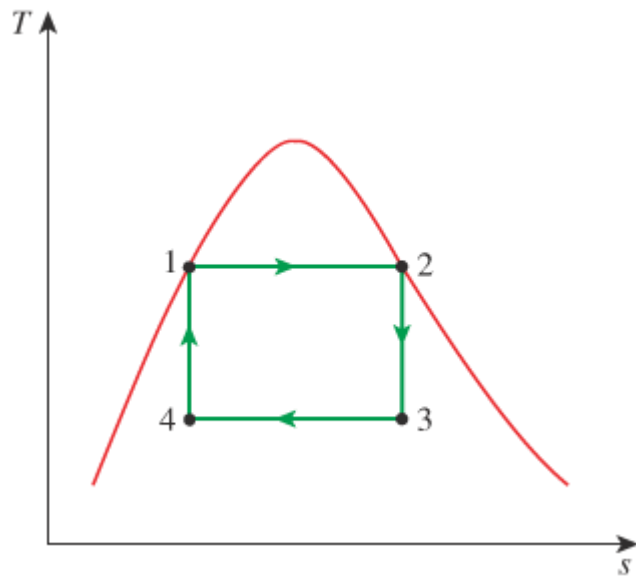
$$F = (\dot{m}V)_{\text{exit}} - (\dot{m}V)_{\text{inlet}} = \dot{m}(V_{\text{exit}} - V_{\text{inlet}}) \quad (\text{N})$$

$$\dot{W}_P = FV_{\text{aircraft}} = \dot{m}(V_{\text{exit}} - V_{\text{inlet}})V_{\text{aircraft}} \quad (\text{kW})$$

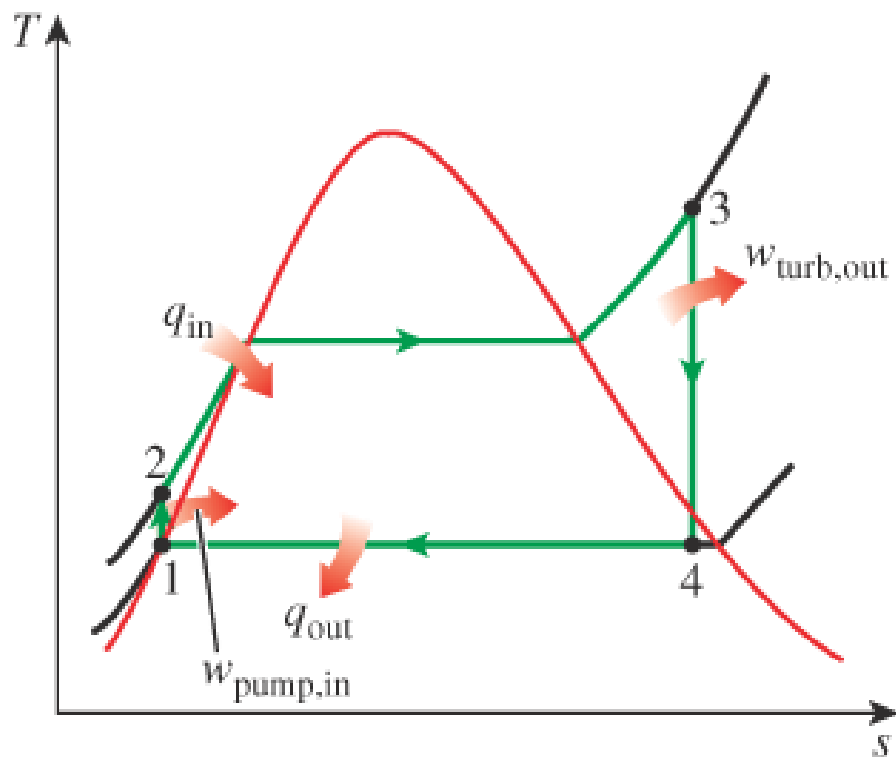
$$\eta_P = \frac{\text{Propulsive power}}{\text{Energy input rate}} = \frac{\dot{W}_P}{\dot{Q}_{\text{in}}}$$

Chapter 10

Carnot vapor cycle



Rankine cycle



Pump ($q = 0$): $w_{\text{pump,in}} = h_2 - h_1$

or,

$$w_{\text{pump,in}} = v(P_2 - P_1)$$

where

$$h_1 = h_{f @ P_1} \quad \text{and} \quad v \cong v_1 = v_{f @ P_1}$$

Boiler ($w = 0$): $q_{\text{in}} = h_3 - h_2$

Turbine ($q = 0$): $w_{\text{turb,out}} = h_3 - h_4$

Condenser ($w = 0$): $q_{\text{out}} = h_4 - h_1$

$$\eta_{\text{th}} = \frac{w_{\text{net}}}{q_{\text{in}}} = 1 - \frac{q_{\text{out}}}{q_{\text{in}}}$$

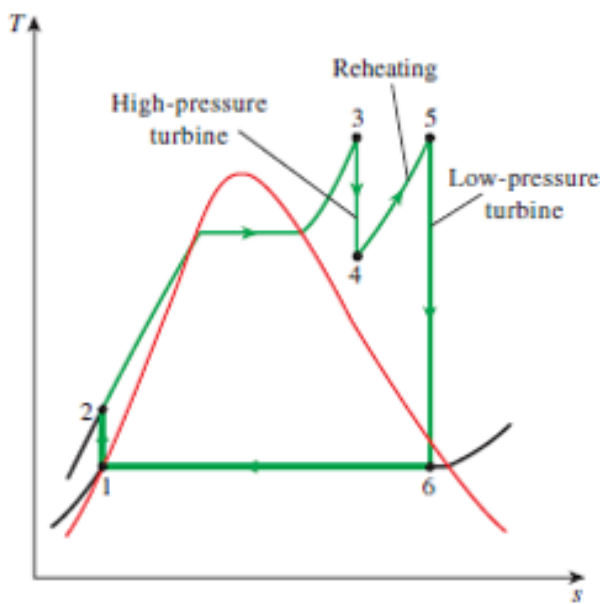
$$w_{\text{net}} = q_{\text{in}} - q_{\text{out}} = w_{\text{turb,out}} - w_{\text{pump,in}}$$

$$\eta_{\text{th}} = \frac{3412 \text{ (Btu/kWh)}}{\text{Heat rate (Btu/kWh)}}$$

$$\eta_P = \frac{w_s}{w_a} = \frac{h_{2s} - h_1}{h_{2a} - h_1}$$

$$\eta_T = \frac{w_a}{w_s} = \frac{h_3 - h_{4a}}{h_3 - h_{4s}}$$

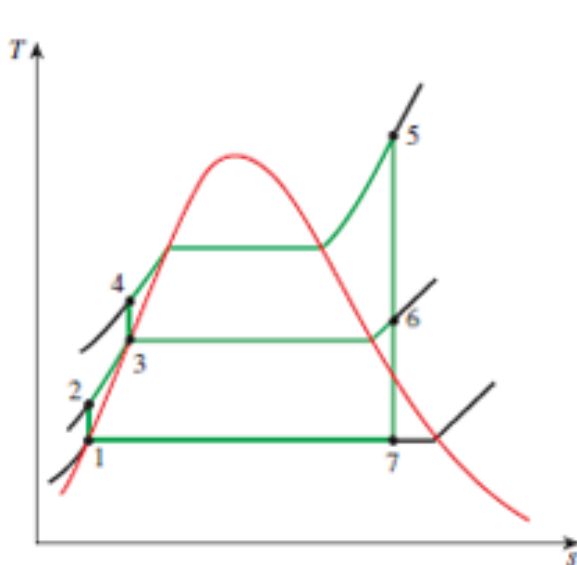
The ideal reheat Rankine cycle



$$q_{in} = q_{primary} + q_{reheat} = (h_3 - h_2) + (h_5 - h_4)$$

$$w_{turb,out} = w_{turb,I} + w_{turb,II} = (h_3 - h_4) + (h_5 - h_6)$$

The ideal regenerative Rankine cycle with an open feedwater heater



$$q_{in} = h_5 - h_4$$

$$q_{out} = (1 - y)(h_7 - h_1)$$

$$w_{turb,out} = (h_5 - h_6) + (1 - y)(h_6 - h_7)$$

$$w_{pump,in} = (1 - y)w_{pump I,in} + w_{pump II,in}$$

$$y = \dot{m}_6 / \dot{m}_5 \quad (\text{fraction of steam extracted})$$

$$w_{pump I,in} = v_1(P_2 - P_1)$$

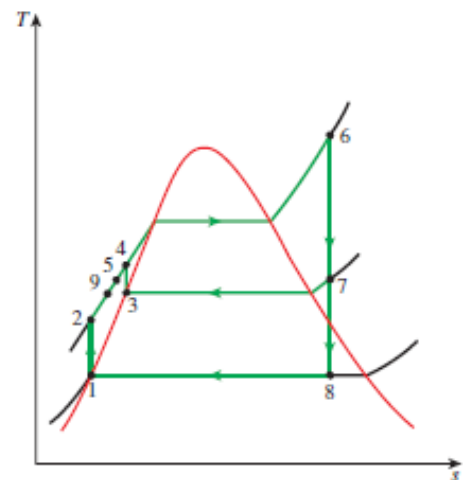
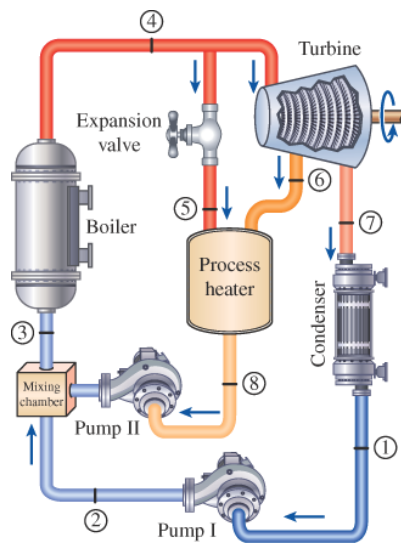
$$w_{pump II,in} = v_3(P_4 - P_3)$$

$$\dot{E}_{in} = \dot{E}_{out} \longrightarrow \sum_{in} \dot{m}h = \sum_{out} \dot{m}h$$

$$yh_6 + (1 - y)h_2 = 1(h_3)$$

$$y = \frac{h_3 - h_2}{h_6 - h_2}$$

The ideal regenerative Rankine cycle with a closed feedwater heater



Cogeneration

$$\dot{Q}_{\text{in}} = \dot{m}_3(h_4 - h_3)$$

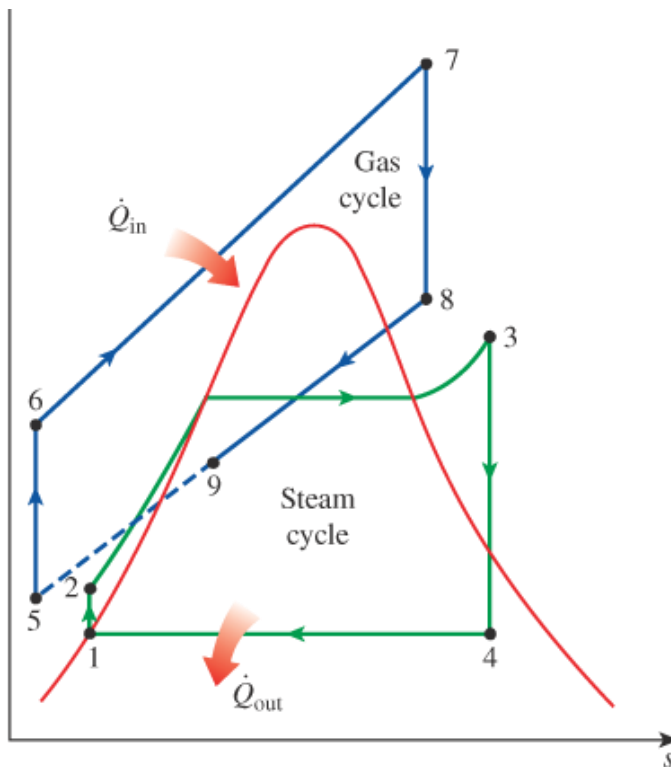
$$\dot{Q}_{\text{out}} = \dot{m}_7(h_7 - h_1)$$

$$\dot{Q}_p = \dot{m}_5 h_5 + \dot{m}_6 h_6 - \dot{m}_8 h_8$$

$$\dot{W}_{\text{turb}} = (\dot{m}_4 - \dot{m}_5)(h_4 - h_6) + \dot{m}_7(h_6 - h_7)$$

$$\epsilon_u = \frac{\text{Net power output} + \text{Process heat delivered}}{\text{Total heat input}} = \frac{\dot{W}_{\text{net}} + \dot{Q}_p}{\dot{Q}_{\text{in}}}$$

Combined gas-vapor power cycle

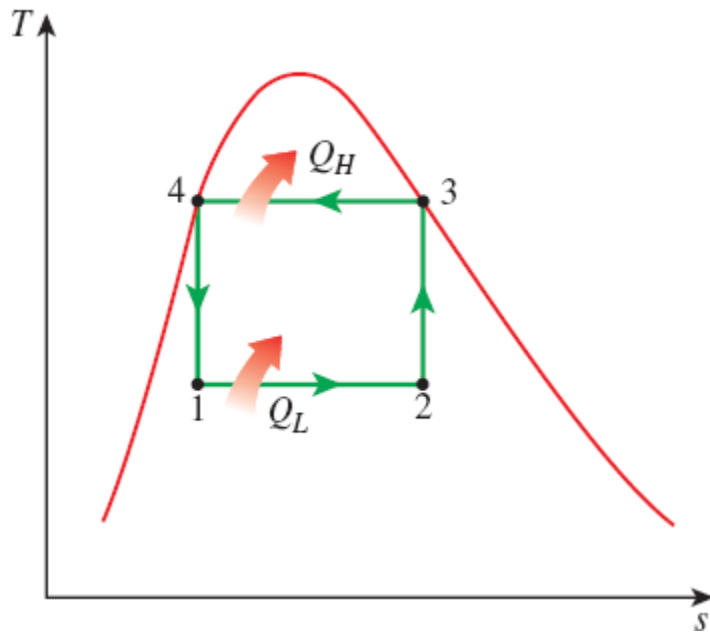


Chapter 11

$$\text{COP}_R = \frac{\text{Desired output}}{\text{Required input}} = \frac{\text{Cooling effect}}{\text{Work input}} = \frac{Q_L}{W_{\text{net,in}}}$$

$$\text{COP}_{\text{HP}} = \frac{\text{Desired output}}{\text{Required input}} = \frac{\text{Heating effect}}{\text{Work input}} = \frac{Q_H}{W_{\text{net,in}}}$$

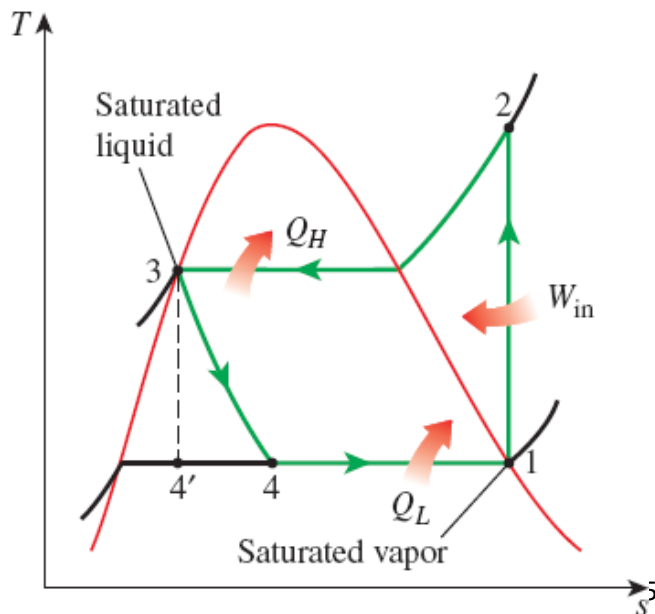
Reversed Carnot cycle



$$\text{COP}_{R,\text{Carnot}} = \frac{1}{T_H/T_L - 1}$$

$$\text{COP}_{\text{HP,Carnot}} = \frac{1}{1 - T_L/T_H}$$

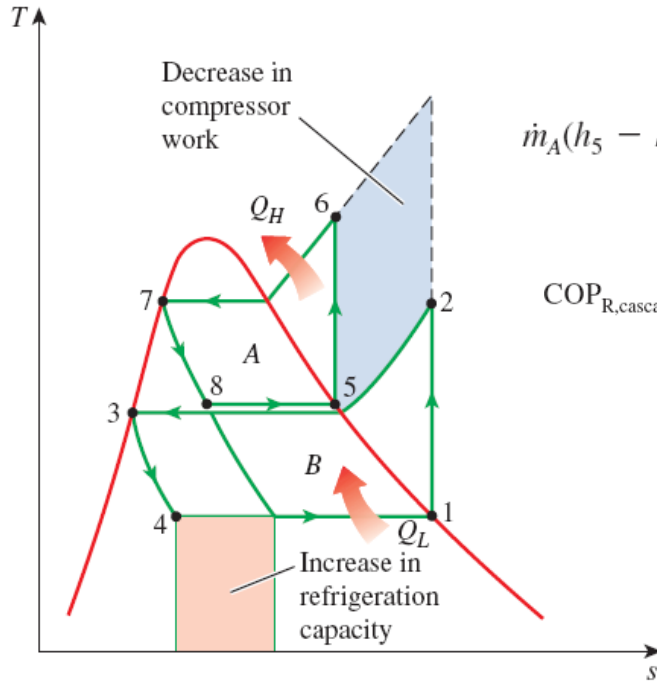
The ideal vapor-compression refrigeration cycle



$$\text{COP}_R = \frac{q_L}{w_{\text{net,in}}} = \frac{h_1 - h_4}{h_2 - h_1}$$

$$\text{COP}_{\text{HP}} = \frac{q_H}{w_{\text{net,in}}} = \frac{h_2 - h_3}{h_2 - h_1}$$

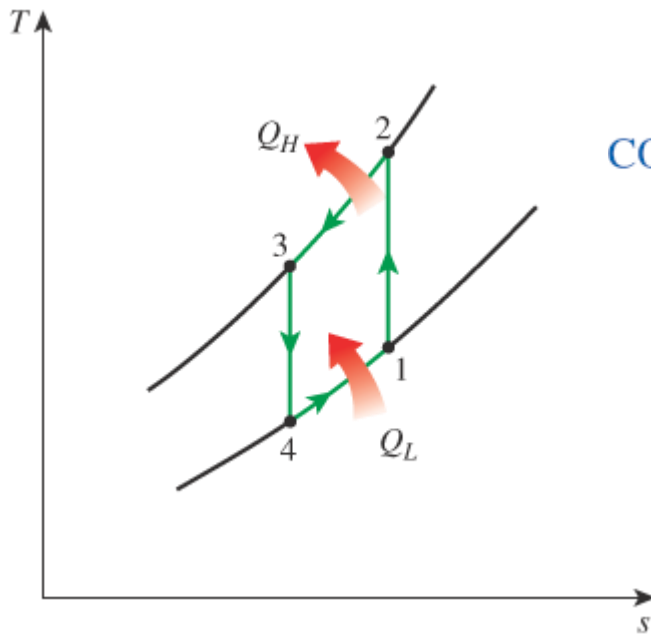
Cascade refrigeration cycles



$$\dot{m}_A(h_5 - h_8) = \dot{m}_B(h_2 - h_3) \longrightarrow \frac{\dot{m}_A}{\dot{m}_B} = \frac{h_2 - h_3}{h_5 - h_8}$$

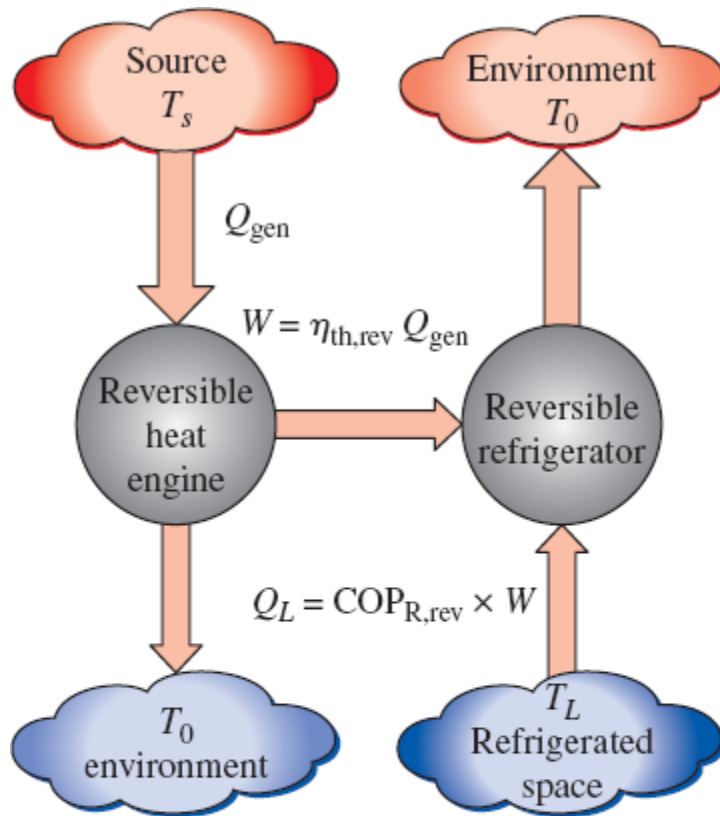
$$\text{COP}_{R,\text{cascade}} = \frac{\dot{Q}_L}{\dot{W}_{\text{net,in}}} = \frac{\dot{m}_B(h_1 - h_4)}{\dot{m}_A(h_6 - h_5) + \dot{m}_B(h_2 - h_1)}$$

Gas refrigeration cycles



$$\text{COP}_R = \frac{q_L}{w_{\text{net,in}}} = \frac{q_L}{w_{\text{comp,in}} - w_{\text{turb,out}}}$$

Absorption refrigeration cycles



$$W = \eta_{\text{th, rev}} Q_{\text{gen}} = \left(1 - \frac{T_0}{T_s}\right) Q_{\text{gen}}$$

$$Q_L = \text{COP}_{\text{R,rev}} W = \left(\frac{T_L}{T_0 - T_L}\right) W$$

$$\text{COP}_{\text{rev,absorption}} = \frac{Q_L}{Q_{\text{gen}}} = \left(1 - \frac{T_0}{T_s}\right) \left(\frac{T_L}{T_0 - T_L}\right)$$

$$\text{COP}_{\text{absorption}} = \frac{\text{Desired output}}{\text{Required input}} = \frac{Q_L}{Q_{\text{gen}} + W_{\text{pump}}} \cong \frac{Q_L}{Q_{\text{gen}}}$$

$$\text{COP}_{\text{rev,absorption}} = \frac{Q_L}{Q_{\text{gen}}} = \eta_{\text{th,rev}} \text{COP}_{\text{R,rev}} = \left(1 - \frac{T_0}{T_s}\right) \left(\frac{T_L}{T_0 - T_L}\right)$$